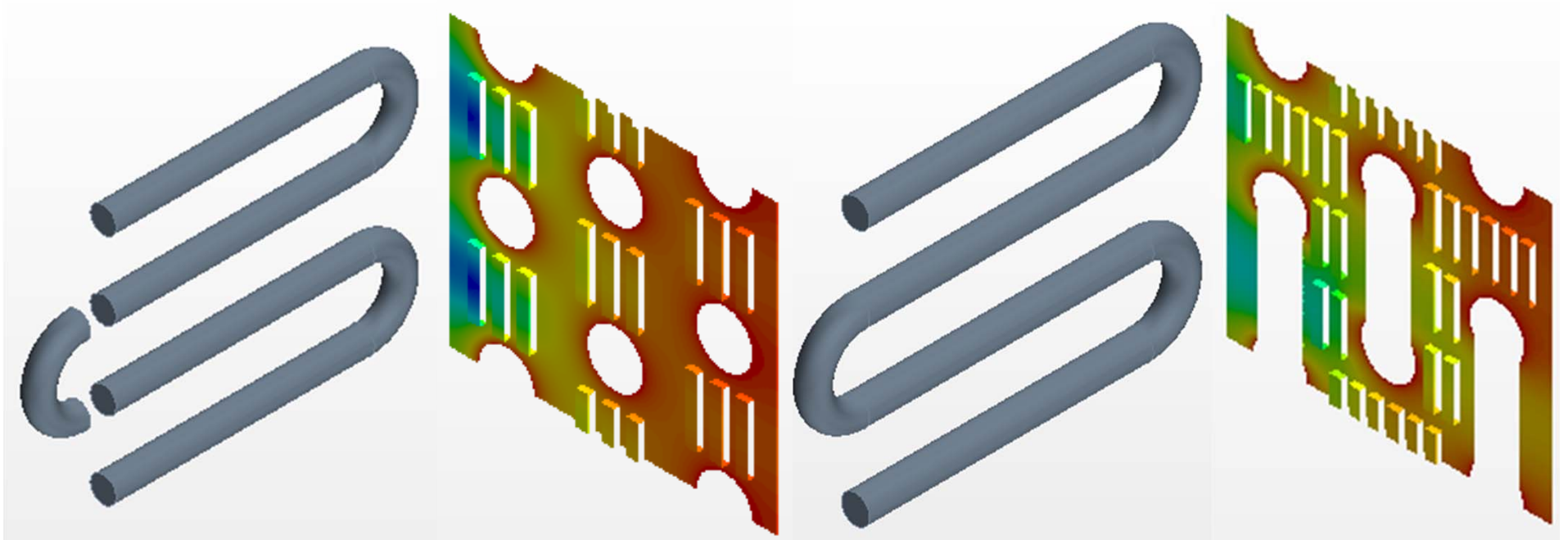


# Advanced Serpentine Heat Exchangers

2017 Building Technologies Office Peer Review



U.S. DEPARTMENT OF  
**ENERGY**

Energy Efficiency &  
Renewable Energy

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# Project Summary

## Timeline:

Start date: 10/2016

Planned end date: 10/2019

## Key Milestones

1. Develop Optimized Fin Geometry; 08/2017
2. Construct Prototype Heat Exchangers; 03/2018
3. Commercialization Plan; 10/2019

## Budget:

**Total Project \$ to Date: 23,937.97**

- DOE: \$17,884.40
- Cost Share: \$6,053.57

**Total Project \$: 663,397**

- DOE: \$509,563
- Cost Share: \$153,834

## Key Partners:

Optimized Thermal Systems, Inc.
Heat Transfer Technologies (HTT)
United Technologies Research Center (UTRC)

## Project Outcome:

Conceptualize serpentine heat exchangers for HVAC application, aiming leakage reduction.

Design & Optimize novel “dog-bone” fin concepts that result in equivalent or better performance than current state-of-the-art tube-fin heat exchangers.

Prototype, validate and commercialize.

# Purpose and Objectives

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**Problem Statement:** develop heat exchanger design and manufacturing solutions aiming to significantly reduce the direct and indirect impacts of refrigerant leakage in heating, ventilating, air conditioning and refrigeration (HVAC&R) systems.

## Main Objectives:

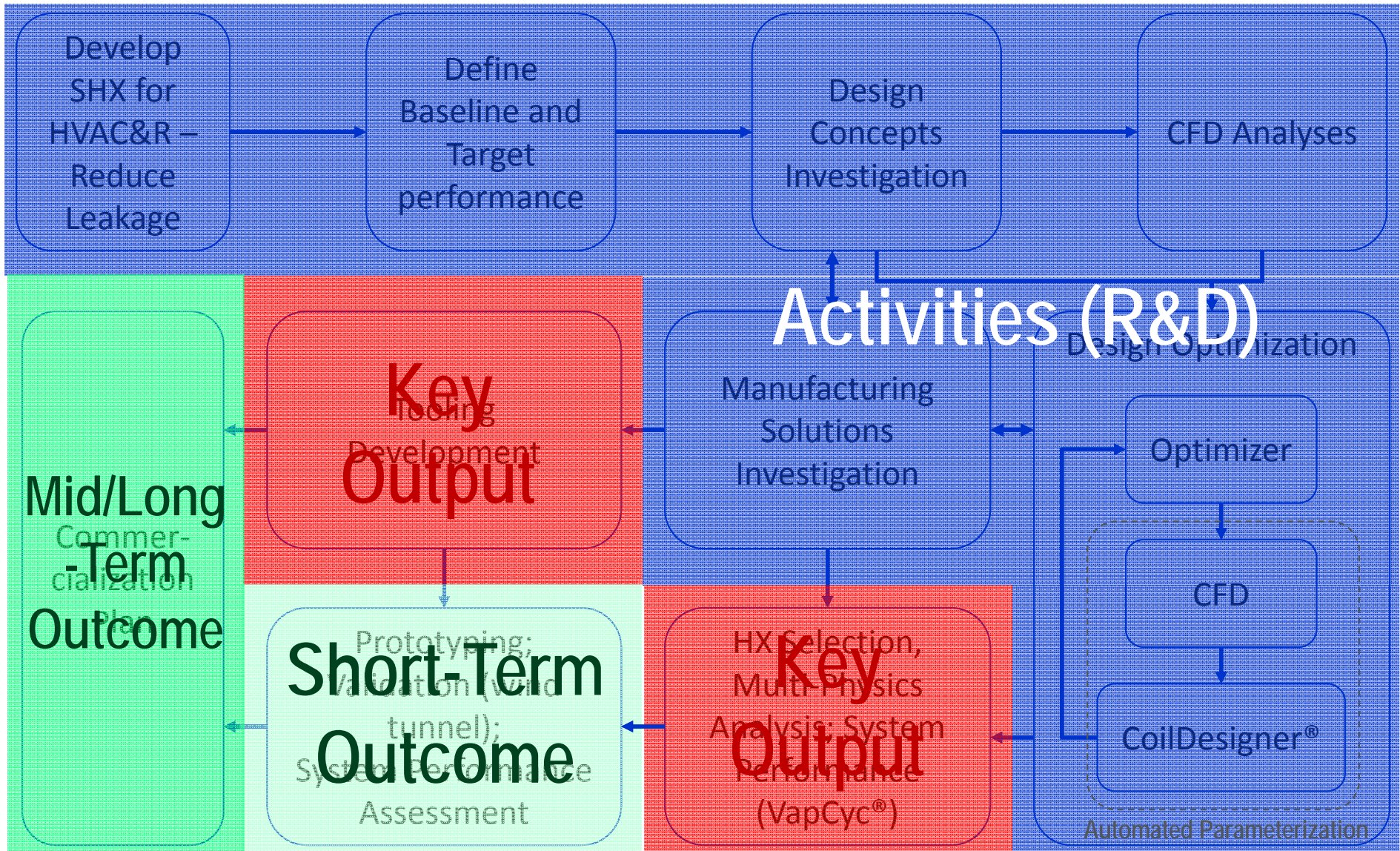
- Develop serpentine heat exchangers (SHX) with equivalent or better performance than current state-of-the-art HX's
- Eliminate 90% of the joints in a 3-ton residential AC / heat pump system
- Develop designs that could be mass produced

**Target Market and Audience:** Heat Pumps and Air Conditioners – **31%** of end energy use in residential and commercial buildings in 2014

## Impact of Project:

- a) New SHX concepts with reduced leakage potential – **Year 1**
- b) New tooling and manufacturing approaches – **Years 1, 2**
- c) SHX Design & Optimization; Prototype & Validation – **Year 2**
- d) Commercialization Plan – **Year 3**
- e) Heat Pumps with reduced leakage potential – 2-3 Years after project

# Approach

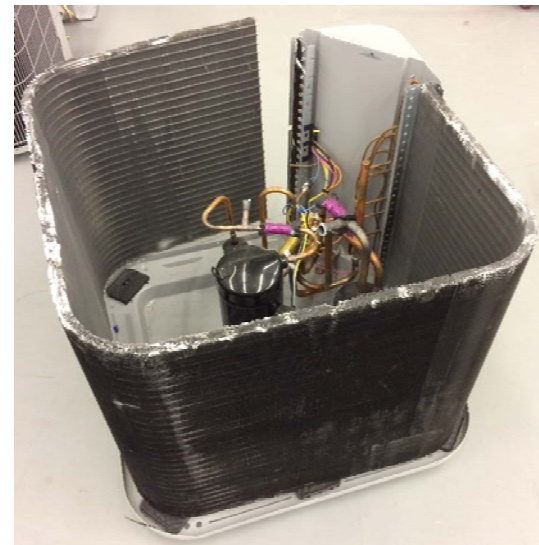
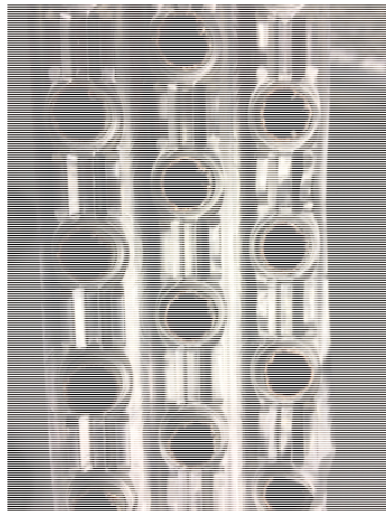
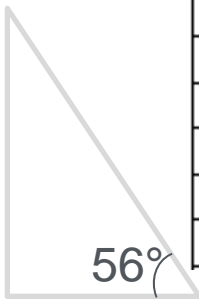


# Progress and Accomplishments - Tasks & Milestones

Task	Milestone	Time frame	Status
<b>Task 1.1:</b> Baseline Selection	<b>Milestone 1.1:</b> Provide details of selected baseline systems and heat exchangers and provide rationale for selection	10/16 – 11/16	<ul style="list-style-type: none"> <li>✓ Baseline selected: Carrier state-of-the-art 3Ton unit</li> <li>✓ CoilDesigner® verification</li> </ul>
<b>Task 1.2:</b> Initial Performance Simulations.	<b>Milestone 1.2:</b> Demonstrate that a non-optimized enhanced “dog-bone” type fin can achieve heat transfer and pressure drop performance within <b>20%</b> of the baseline fin through CFD simulations.	11/16 - 02/17	<ul style="list-style-type: none"> <li>✓ CoilDesigner® optimization studies</li> <li>✓ Challenges identification</li> <li>✓ Baseline re-designed with “dog-bone” fin + CFD simulations</li> <li>✓ CoilDesigner® verification → <b>&lt;20% of baseline performance</b></li> </ul>

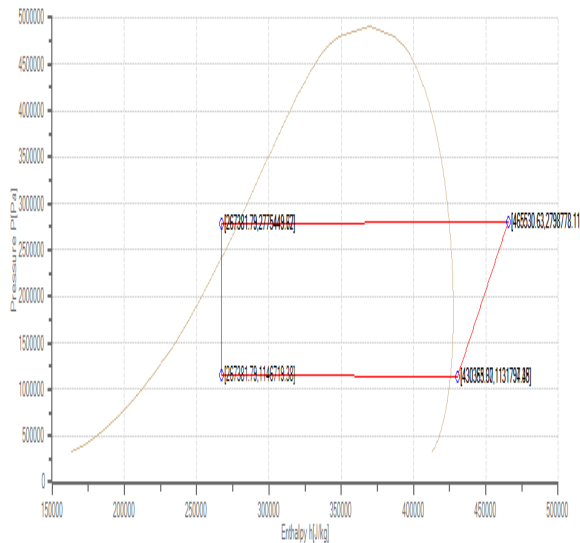
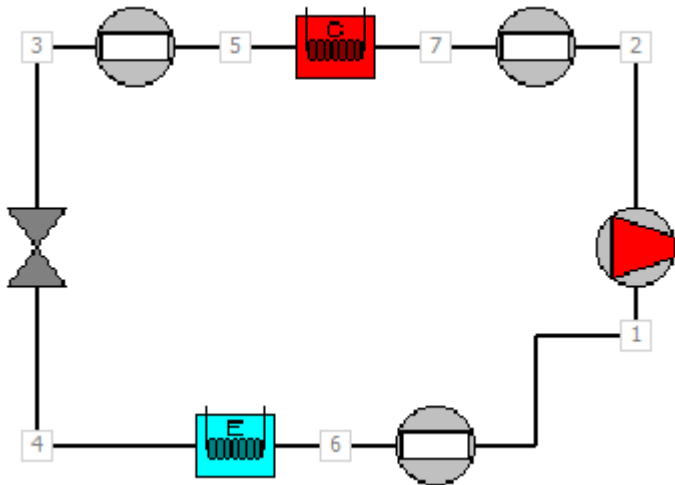
# Baseline Selection

Outdoor HX tube diameter	7mm (0.291" expanded OD)
Outdoor HX # tubes	28
Outdoor HX fin type	Lanced
Outdoor HX fin pattern and geometry details*	0.85" X 0.736"
Outdoor HX air flow rate	3167 CFM
Indoor HX tube diameter	3/8" (0.396" expanded OD)
Indoor HX # tubes	24
Indoor HX fin type	Lanced
Indoor HX fin pattern and geometry details*	1" X 0.75"
Indoor HX air flow rate	1200 SCFM
Compressor model number	APGo31KA



Inner grooved

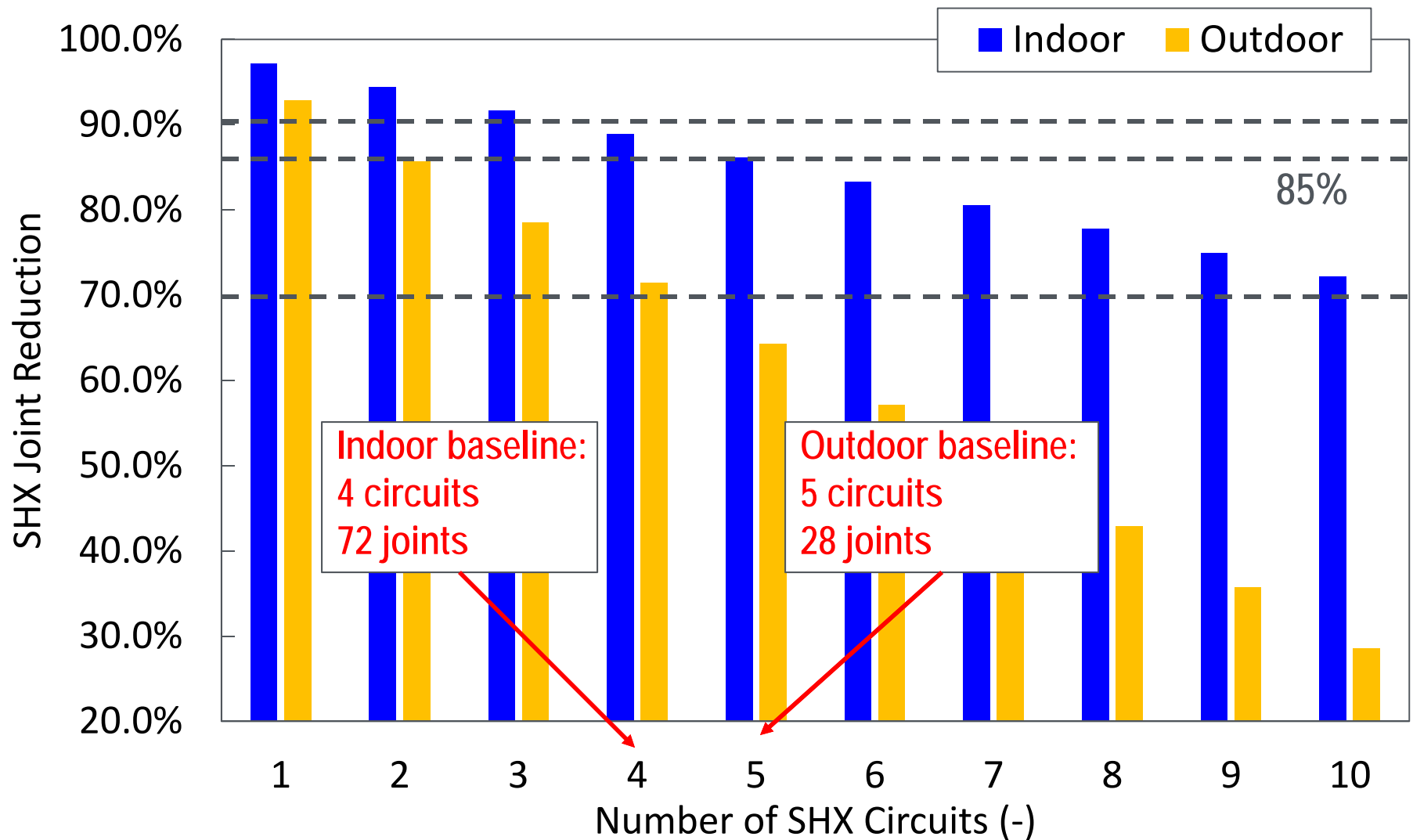
# Baseline Selection – Performance Assessment



VapCyc® System Results		
System COP	4.6	-
Power Consumption	2234	W
Refrigerant Charge	1.48	kg
Subcooling	5.5	K
Superheat	4.0	K

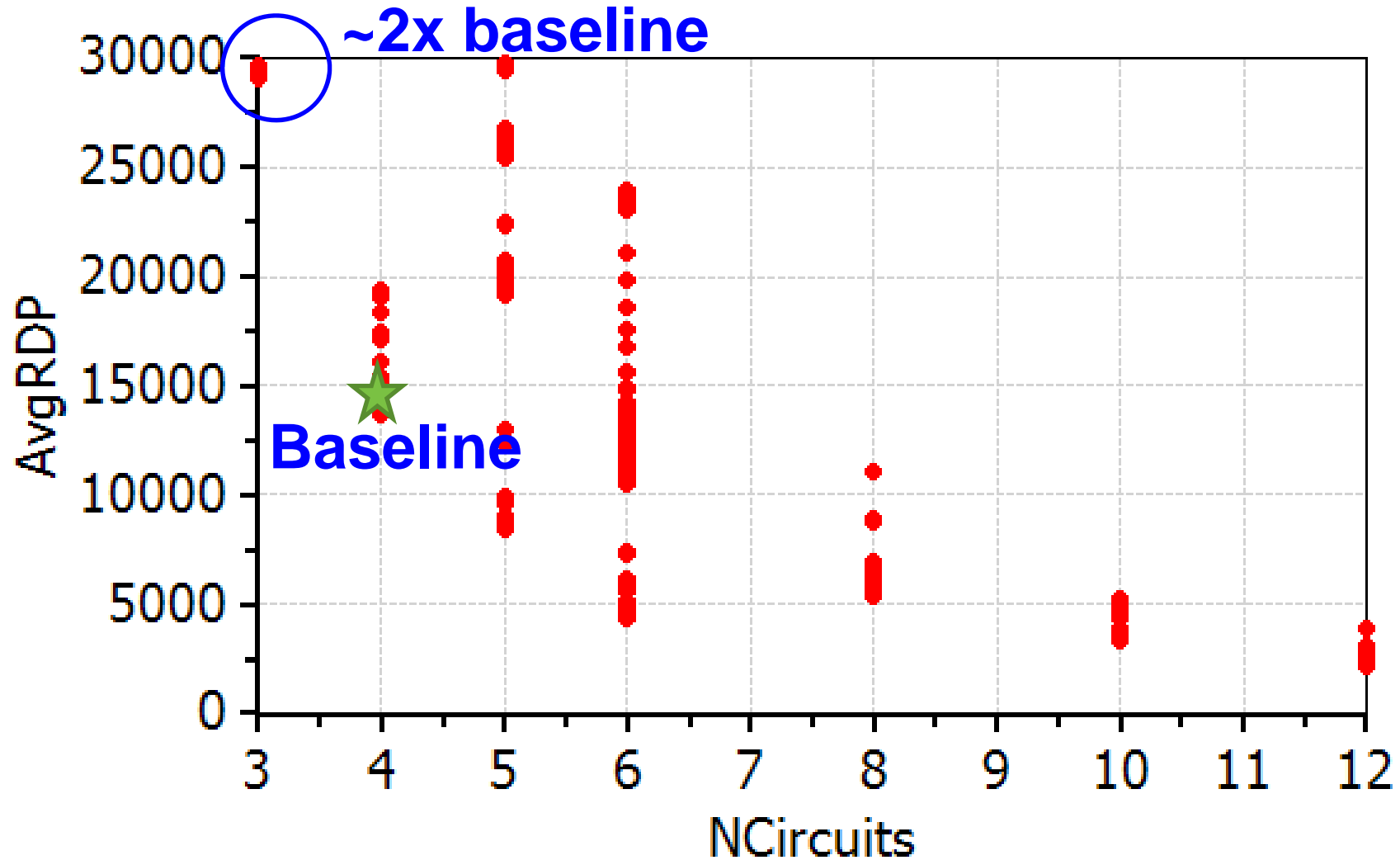
CoilDesigner® Heat Exchanger Results	Unit	Baseline	
		Indoor HX	Outdoor HX
Capacity	kW	10.31	12.58
Air flow rate	cfm	1200	3149
Pressure	kPa	1147	2798
Inlet Temperature	K	285	345
Inlet Quality	-	0.228	-
Air Pressure Drop	Pa	45.8	12.2
Air Heat Transfer Coefficient	W/m <sup>2</sup> .K	101.3	108.1
Refrigerant Pressure Drop	kPa	14.9	23.3

# Initial Analysis – Circuiting vs. Joint Reduction



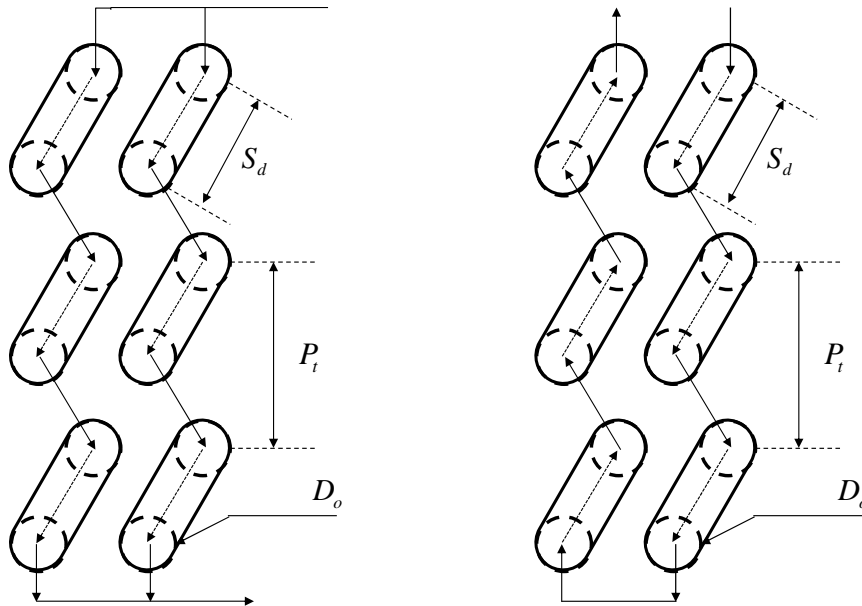


# Initial Analysis - Ref. $\Delta P$ vs. N° circuits (Indoor HX)

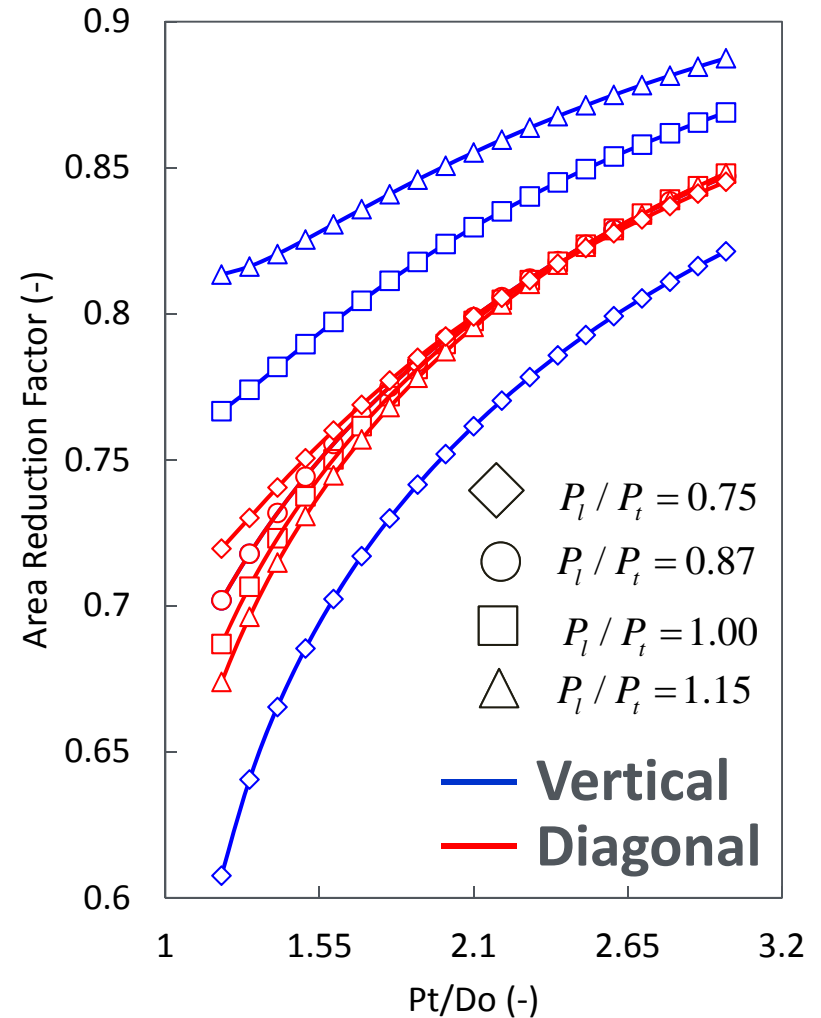
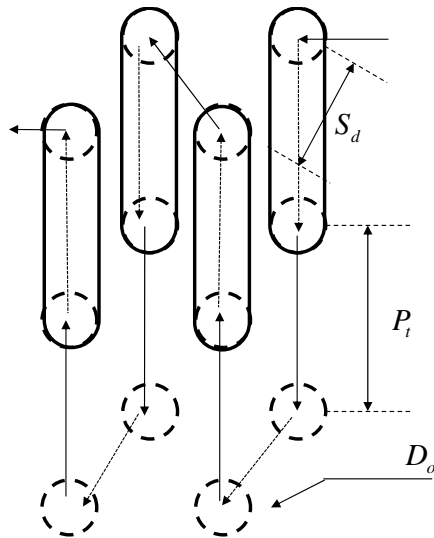


# Initial Analysis – Fin “dog-bone” cut (Indoor HX)

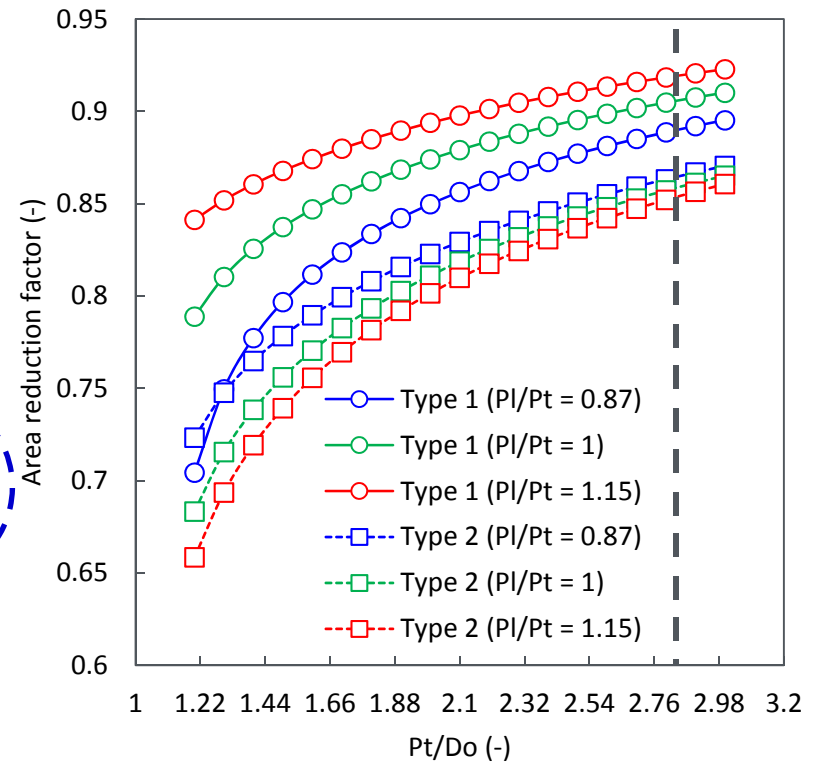
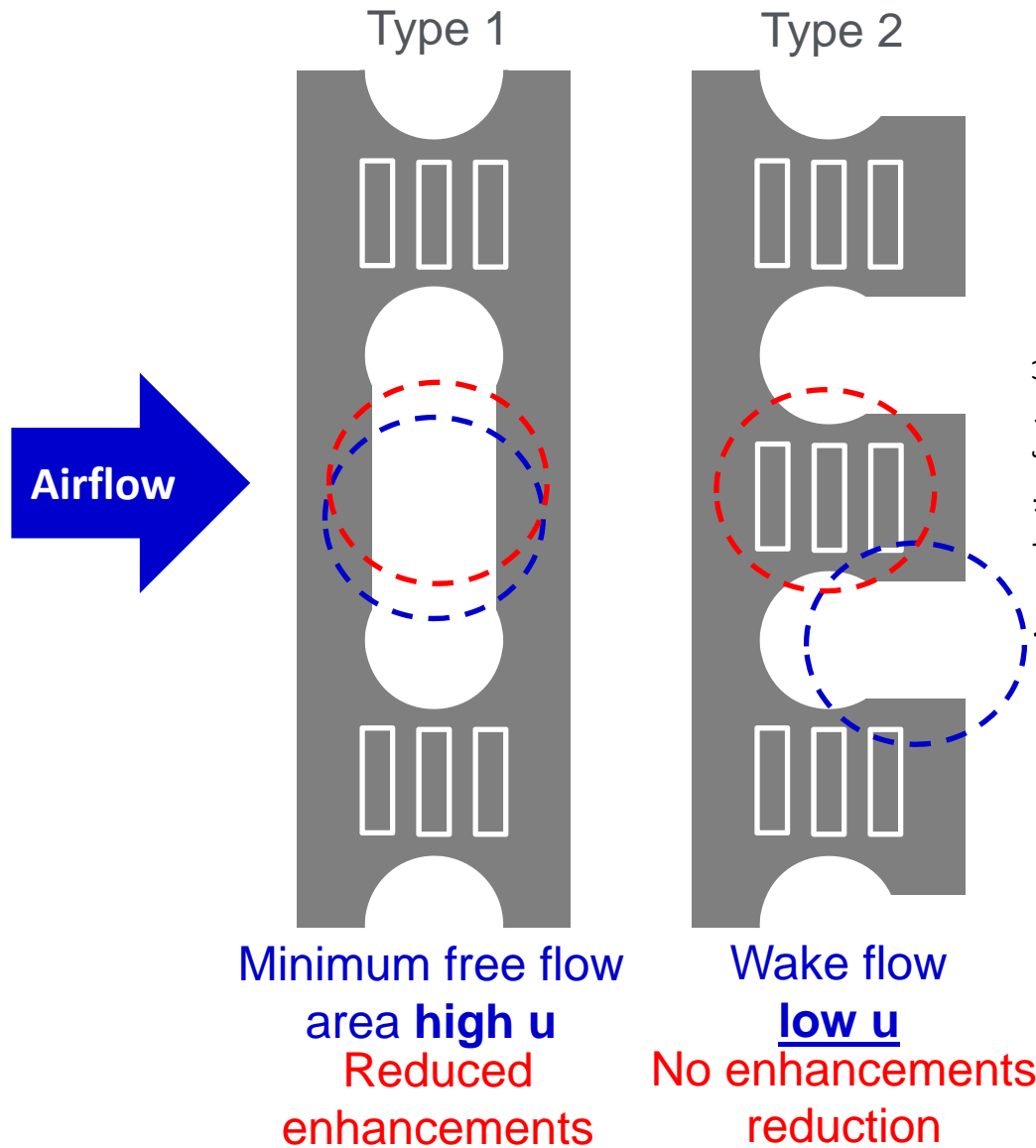
Semi-cross Parallel    Semi-cross Counter



Full-cross Counter



# Initial Analysis – Fin “dog-bone” cut (Outdoor HX)



# Initial Analysis – Manufacturing Considerations

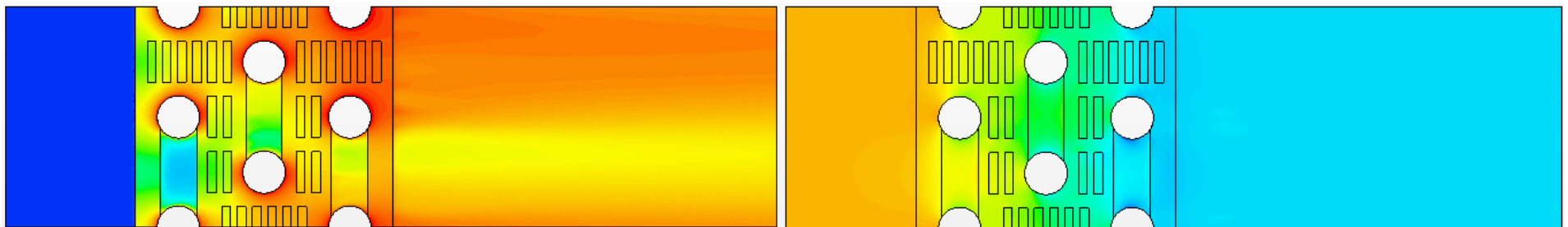
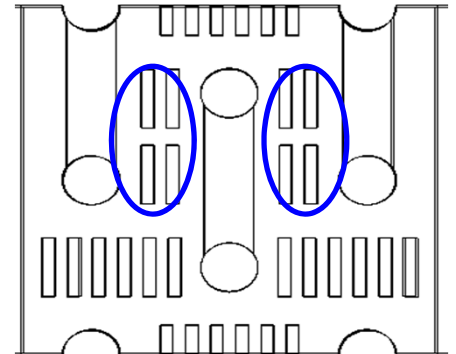
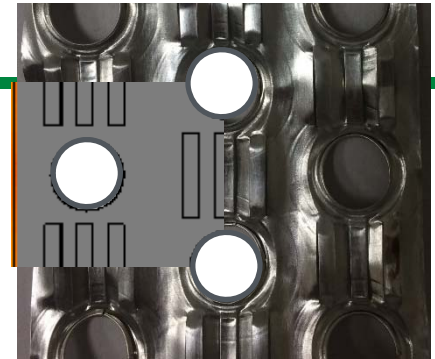
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- Tubes
  - Minimum diameter: 5.0mm
  - Minimum bend radius (Do): 1.5
  - Tube length: N/A
  - Pitch angle: N/A
- Fins
  - Minimum tube-fin enhancement distance: 2.0mm
  - Max fin density: ~20 FPI
  - Assembly could potentially damage enhancements; yet to be confirmed

# Baseline Modification – Indoor HX

## CFD Settings

- Mesh size: 1.0mm (polyhedral elements)
- Boundary layer: 3.0mm, 10 layers, 1.2 growth
- Uncertainty analysis - GCI (on going task)
- Dry air – ideal gas
- K-e Realizable turbulence model
- B. C.: uniform inlet velocity/temperature; constant wall temp.
- Steady-State



Y  
Z X

Temperature (K)  
299.04 309.23 319.42 329.61 339.81 350.00

Y  
Z X

Pressure (Pa)  
-15,000 0.0000 15,000 30,000 45,000 60,000

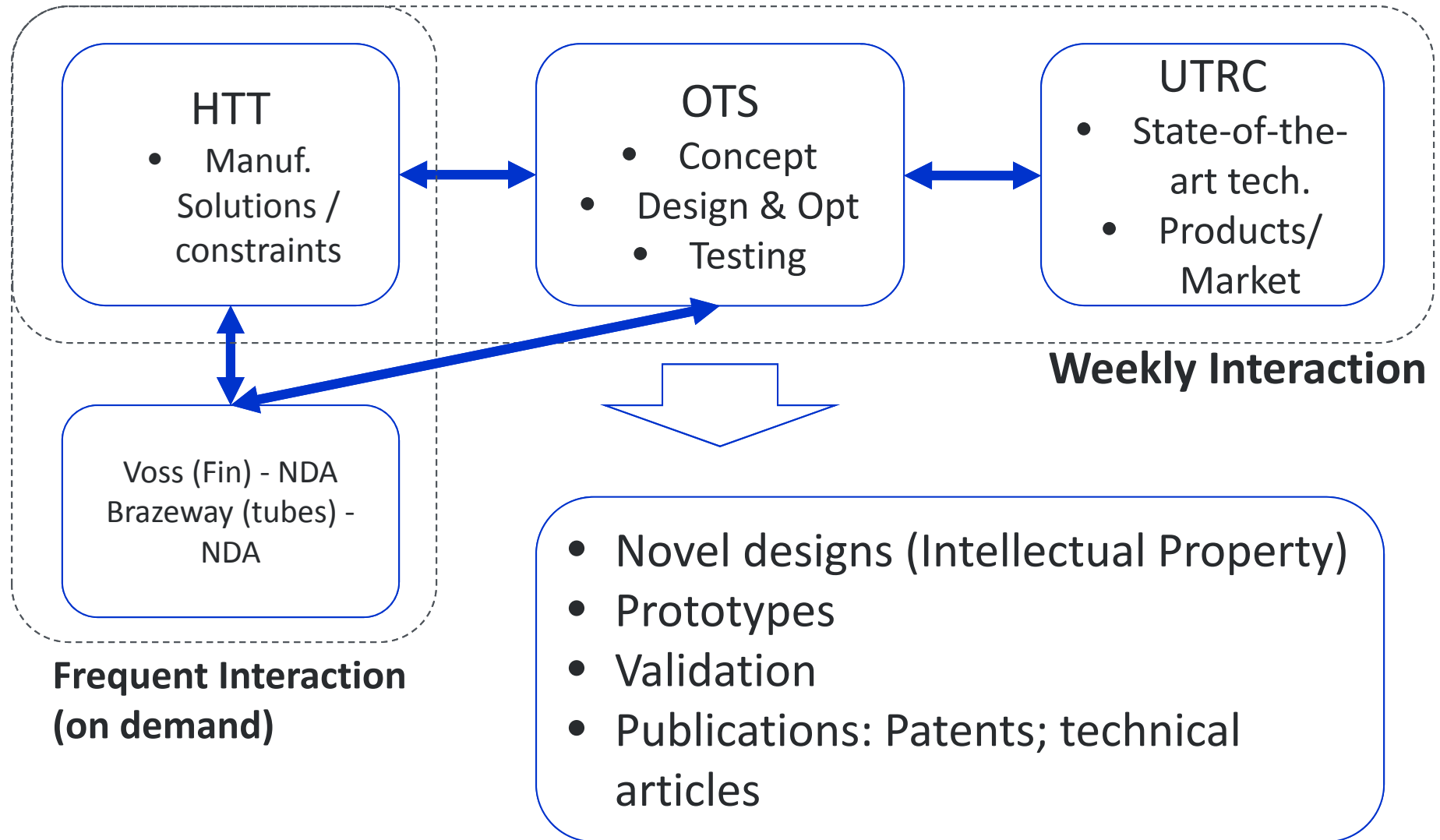
# Milestone 1.2 Accomplishment – Indoor HX

Metric	Unit	Baseline	SHX-POF-001	Rel Diff
Do	m	0.0098	0.0098	0.00%
PI	m	0.019	0.0198	4.21%
Pt	m	0.0254	0.0254	0.00%
Sd	m	0.0229	0.0235	2.93%
FPI	m	16	16	0.00%
Nbanks	-	3	3	0.00%
Nrows	-	24	24	0.00%
Ncircuits	-	4	4	0.00%
Face area	m <sup>2</sup>	0.263	0.263	0.0%
Joints	-	72	8	-88.89%
hair	W/m <sup>2</sup> .K	101.1*	113.3**	12.07%
Ao	m <sup>2</sup>	16.85	14.24	-15.47%
ΔPair	Pa	45.82*	49.12**	8.45%
ΔPref	kPa	14925	14661	-1.77%
Q	W	10351	10187	-1.58%

< 20%

\*Wang et al. (2001) \*\*CFD

# Project Integration and Collaboration



# Next Steps and Future Plans

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- **Task 1.2: Initial Performance Analysis**
  - Finalize CFD analyses (uncertainty analysis; outdoor HX)
- **Task 1.3: Material Simulation and Selection**
  - Study other modifications to the baseline to make it manufacturable
  - Indoor coil: review copper and aluminum options
  - Outdoor coil: exploring options (Task 1.2)
  - HTT begins exploring material and manufacturing options
- **Task 1.4: Benchtop Testing of Brazing Methods (HTT)**



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# REFERENCE SLIDES

# Project Budget

Budget History					
10/2016 – FY 2017		FY 2018		FY 2019 – 10/2019	
DOE	Cost-share	DOE	Cost-share	DOE	Cost-share
\$100,432	\$25,297	\$253,488	\$68,230	\$155,643	\$60,307

# Project Plan and Schedule

Project Schedule												
Project Start: 10/2016	Completed Work											
Projected End: 10/2019	Active Task (in progress work)											
	◆ Milestone/Deliverable (Originally Planned)											
	◆ Milestone/Deliverable (Actual)											
	FY2017				FY2018				FY2019			
Task	Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)	Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)	Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)
<b>Past Work</b>												
1.0 Intellectual Property (IP) Management Plan	◆											
2.1 Baseline Selection	◆											
2.2 Initial Performance Simulations		◆										
<b>Current/Future Work</b>												
2.3 Material Simulation and Selection		◆										
2.4 Benchtop Testing of Brazing Methods			◆									
3.1 Optimization Definition and Manufacturing Considerations				◆								
3.2 Develop Optimized Fin Geometry					◆							
4.1 Design Fin Tooling						◆						
4.2 Construct Prototype Heat Exchangers							◆					
5.1 Heat Exchanger Performance Testing								◆				
5.2 Mechanical / Cyclic Testing									◆			
6.1 Improve Manufacturing Techniques in Preparation for Commercialization										◆		
6.2 System Level Integration											◆	
6.3 System Level Testing												◆
7.0 Develop Technology to Market Commercialization Plan												◆