# **High-Efficiency Commercial Cold Climate Heat Pump**

2016 Building Technologies Office Peer Review





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# **Project Summary**

#### Timeline:

Start date: March 1, 2013

Planned end date: May 29, 2015

#### **Key Milestones (SOPO)**

- 2/ 2013: Down-selection of key components that meet DOE capacity and COP targets (COP=2.5 at -13F ambient condition) through modeling.
- 2. 8/2014 Experimental testing of 1<sup>st</sup> prototype CCCHP system meets (within 10% relative) targeted COP=2.5 at -13F design point.
- 3. 5/ 2015 TRL5 demonstration of 2<sup>nd</sup> prototype CCCHP system and TRL6 plan

#### **<u>Budget</u>**: Project Complete

Total DOE \$ - Final: \$1.5M

Total CS \$ - Final: \$375k

Total future DOE \$: \$0k

Total future CS \$: \$0k

#### **Target Market/Audience**:

Commercial building owners with a need for a superior heating and cooling solution air-source heat pump that operates over extreme heating and cooling seasons

#### **Project Goal**:

- Design and develop a prototype 10 TR high performance cold climate commercial heat pump system
- Execute a Technology Readiness Level (TRL) 5 prototype demonstration
- 3) Conduct psychrometric testing of prototype CCCHP system to demonstrate targeted COP=2.5 at -13F design point with <15% capacity degradation.
- 4) Meet COP and Capacity targets at 47F and 17F ambient conditions
- 5) <3 year customer payback on price premium

## **Purpose and Objectives**

#### **Problem Statement:**

- State-of-the-art industry standard heat pumps that can degrade by up to 60% in capacity and 50% in system Coefficient of Performance (COP) at the DOE-targeted -13F ambient condition
- Cold blow effect further cements that heat pumps are not a viable technology for space heating even in Climate Zone 3A (Memphis TN).
- Scalable and cost-effective compression technologies as well as system optimization/integration are necessary to deploy cold climate heat pumps

**Target Market and Audience**: Commercial building owners with a need for a superior heating and cooling solution air-source HP that operates over extreme heating and cooling seasons. Commercial buildings in cold climates represent 45% of the national building stock. 32% use electricity as the primary source of space heating representing 149 billion kWh and \$9.2 billion.

#### **Impact of Project:**

End deliverable of project: TRL5 demonstration of CCCHP system that realizes >20% annual energy savings

Achievement towards your goal: Intermediate-term (2-3yr after project)



## **Approach**

#### Approach:

- 1) Evaluate and down-select a variable speed high-efficiency compressor concept that meets both performance and cost targets
- 2) Design, develop, and fabricate components that enable the proposed compressor performance
- 3) Design and integrate high efficiency evaporator and condenser fans and motors
- 4) Evaluate whether the specific characteristics of refrigerant fluids can be used to improve system efficiency
- 5) Demonstrate performance of an integrated 10TR CCCHP prototype over a wide range of ambient conditions
- 6) Document CCCHP performance, cost and short-term reliability characteristics.

**Key Issues:** Reduce component and system critical risks to sufficient level to ensure TRL5 demonstration is successful

**Distinctive Characteristics**: Combination of compression with high efficiency over an unusually wide range of speed and pressure ratio, losses reduction and system-level design optimization for cold climates will enable this performance.



## **Lab Capabilities**

This UTRC facility provides the ability to test air-to-refrigerant and hydronic systems under a wide range of operating conditions within world-class energy balances

#### Capability

- Matching 5TR indoor and outdoor rooms
- >10TR indoor and outdoor rooms

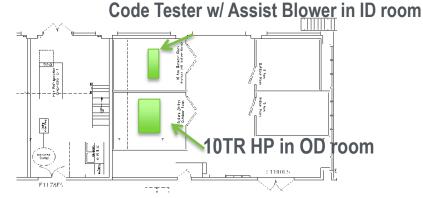
#### Operating Ranges

- Air-side (indoor room): 45 to 145 °F, 30 to 95%RH, and 4,000 SCFM
- Air-side (outdoor rooms): -15 to 145 °F, 20 to 95%RH, and 6,500 SCFM and 8,500 SCFM
- Hydronic-side: 40 to 200 °F (5TR and 10TR)

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	d.
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Test Energy Balance

Ambient	1-Q <sub>air</sub> /Q <sub>refrig</sub>	Typical
47F	4%	~4%
17F	8%	~11%
-13F	<16%	unknown





#### **Lessons Learned:**

- UTRC developed 3 prototype 10TR heat pump systems to optimize architecture necessary to test & demonstrate >20% annual energy savings
- Risk mitigation through sequential hardware demonstrations guided by modeling required to achieve success

#### **Accomplishments:**

**Demonstrated** 

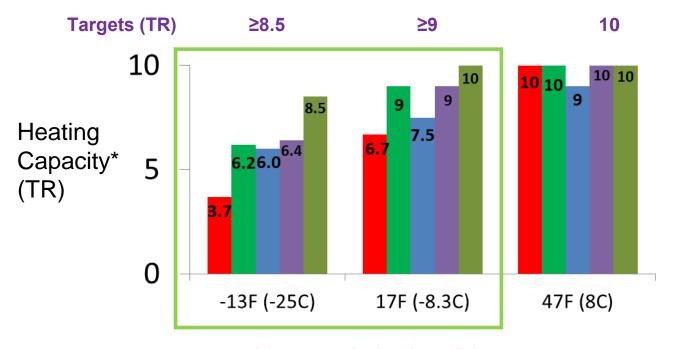
- >130% capacity and >38% COP increase over baseline at -13F condition
- Demonstrated, within accuracy of measurements and conditions, DOE capacity goals at 17F/47F of 9TR/10TR while exceeding baseline COP at 47F
- System and component-level model validation

In pursuit of DOE goals,

High-fidelity test data from POC system (<16% energy balance at -13F design point) to confirm modeling projections associated with the contribution of key enabling technologies



Tested 130% & 50% capacity improvement at -13 & 17F ambient conditions; Capacity at -13 & 17F critical for value proposition



**UTRC CCCHP Prototype System** 



\*Air-side capacity; energy balance quantified Baseline HP : mid-tier high-efficiency 10TR

POC System : wide speed range compressor +reduced

losses and high-efficiency fans

1<sup>st</sup> Prototype : POC System + charge management

2<sup>nd</sup> Prototype :1<sup>st</sup> Prototype + Optimized ID Coil + Optimized OD Coil

3<sup>rd</sup> Prototype : 2<sup>nd</sup> Prototype + optimized compressor



Energy Efficiency & Renewable Energy

Tested >38% COP improvement at -13F ambient; COP at 17F and 47F critical for value proposition



\*Air-side capacity; energy balance quantified

Baseline HP : mid-tier high-efficiency 10TR

POC System : wide speed range compressor +reduced

losses and high-efficiency fans

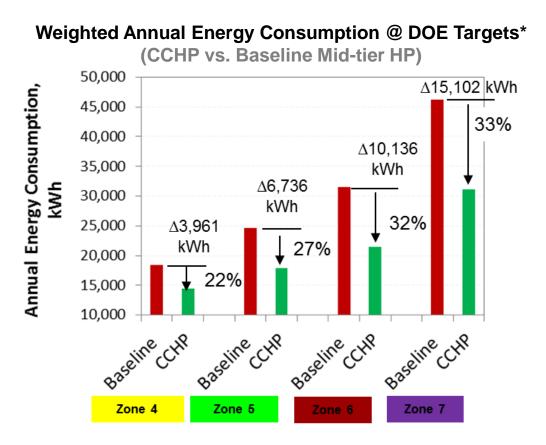
1<sup>st</sup> Prototype : POC System + charge management

2<sup>nd</sup> Prototype :1<sup>st</sup> Prototype + Optimized ID Coil + Optimized OD Coil

3<sup>rd</sup> Prototype : 2<sup>nd</sup> Prototype + optimized compressor



>20% annual energy savings potential demonstrated through building hourly analysis



\*Integrated capacity accounting for frost Non-economized small office building

**Market Impact**: The UTRC team is engaged with the Commercial North **American** HVAC division of **Carrier Corporation**, an operating division of UTC Building and Industrial Systems. This engagement ensures that metrics are met during development to accelerate future transition. Carrier Corporation proprietary tools and other standard work are being used throughout the execution of the project which helps transition the developed system seamlessly.



# **Project Integration and Collaboration**

#### **Project Integration:**

- Carrier Corporation is the commercialization path for HVAC technologies and concepts developed at UTRC.
- Carrier is the world's largest manufacturer and distributor of HVAC&R equipment and has a long history of developing successfully commercialized products.
- UTRC project team is closely engaged with product and engineering teams to ensure metrics are met during conceptualization and testing phases
- Carrier directly providing cost share for this project and significant in-kind contribution

**Partners, Subcontractors, and Collaborators**: UTRC is only performing organization under this contract

Communications: Project is complete; Further evaluation at Carrier



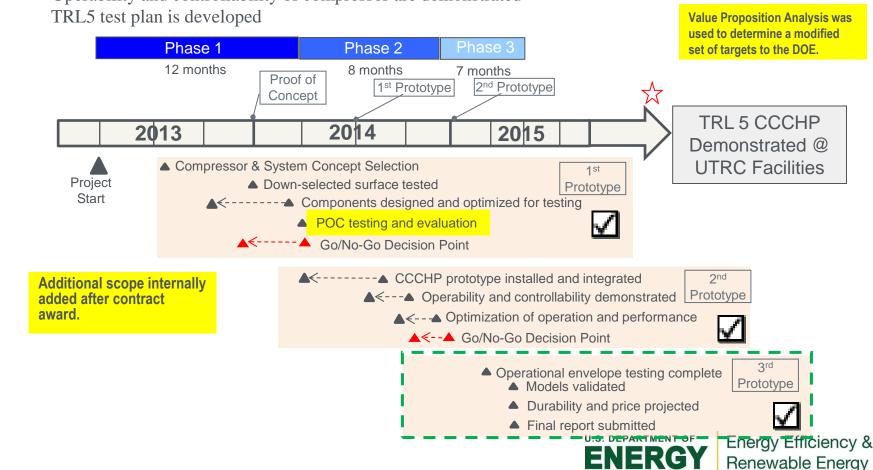
### **Project Timeline**

3 prototype system demonstrations completed on-time and within budget **Phase I GO/NO-GO DECISION:** 

Down-selection of key components that meet (within <10% relative deviation of performance) DOE capacity and COP targets (COP=2.5 at -13F ambient condition) through modeling.

#### Phase II GO/NO-GO DECISION:

- Experimental testing of prototype CCCHP system meets targeted COP=2.5 at -13F design point
- Operability and controllability of compressor are demonstrated



# **Project Budget**

**Project Budget**: See below Table

Variances: None. All UTRC hardware modification and testing is within original

schedule and budget constraints

Cost to Date (5/2015): 100%

Additional Funding: None.

**Schedule:** No changes to original schedule.

Budget History								
FY2013		FY2014		FY2015				
(past)		(past)		(past)				
DOE	Cost-share	DOE	Cost-share	DOE	Cost-share			
\$535,322	\$133,830	\$765,752	\$191,936	\$191,873	\$47,468			



# **Acknowledgments**

This material is based upon work supported by the Department of Energy [Building Technologies Office] under award DE-EE0006108 [High-efficiency Commercial Cold Climate Heat Pump (CCCHP), PI: Dr. A. M. Mahmoud] to the United Technologies Research Center. This presentation was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor any agency thereof, nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, process, or process disclosed, or represents that its use would infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or any agency thereof. The views and opinions of authors expressed herein do not necessarily reflect those of the United States Government or any agencies thereof. The authors are grateful for the support of the Department of Energy (DOE).



# **CCCHP Photographs**



Cut-away view of baseline heat pump system before retrofitting



CCCHP system as installed in Psychrometric chamber at UTRC



Integrated High-efficiency outdoor fans